

NEW THERMAL LINK FINITE ELEMENT FOR FGM CONDUCTORS

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A new effective thermal link finite element is presented in this paper, which is suitable for thermal analysis of the conductors made of a functionally graded material (FGM) with spatial continuous variation of material properties, continuously distributed generated heat along the element length and continuous variation of the convection from the element surface. Numerical experiment will be done concerning calculation of the thermal fields of the FGM conductor. The solution results will be discussed and compared with those obtained using a very fine mesh of the 3D solid finite elements of the FEM program ANSYS.

Keywords: functionally graded material, thermal analysis, heat convection, FGM link finite element, ANSYS, Mathematica

1 INTRODUCTION

This contribution deals with derivation of a new link finite element for thermal analysis including heat generation in the link and heat convection from the element surface. This derivation of the thermal link element is based on exact solution of differential equation for heat transfer. The solution uses new approach in the derivation of FEM equations for link element and also includes heat convection that was not used in our previous works [1, 2]. Numerical experiment will be done in order to compare calculated results with those obtained by simulation using classic FEM elements in program ANSYS.

2 THEORETICAL BACKGROUND FOR HEAT TRANSFER

Let us consider a two nodal straight thermal link element with length L , cross-sectional area $A(x)$ and thermal conductivity $\lambda(x)$. We also consider heat generation $Q(x)$ in the link volume and heat convection $\alpha(x)$ from the link surface. Ambient temperature is T_{ok} and general boundary conditions are denoted as $OP1$ and $OP2$ (Fig. 1).

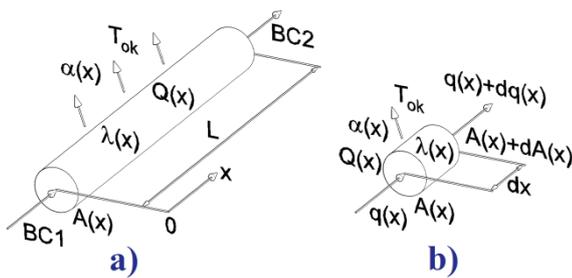


Fig. 1. Thermal link element

According to [3] we can write differential equation for heat transfer

$$-\lambda(x) \frac{d^2 T(x)}{dx^2} - \frac{d\lambda(x)}{dx} \frac{dT(x)}{dx} + \alpha(x) T(x) \frac{o}{A} = Q(x) + \alpha(x) T_{ok} \frac{o}{A} \quad (1)$$

where o is the perimeter of the link.

3 FEM EQUATIONS DERIVATION

Method for solving differential equations with arbitrary order, inconstant coefficients and right side is described in [4]. The method is performed by computer code that does not use high level mathematical operations (like numeric derivation or integration). This method is suitable only for 1D problem.

Let our differential equation has this form

$$\sum_{u=0}^m \eta_u(x) y^{(u)}(x) = \sum_{j=0}^g q_j a_j(x) \quad (2)$$

where, considering also (1), we can write:

- $m = 2$ order of the differential equation
- $y(x) \equiv T(x)$ unknown function of x is the temperature function
- $y^{(u)}(x) \equiv \frac{d^u T(x)}{dx^u}$ u^{th} derivation of unknown function
- $\eta_0(x) = \alpha(x) \frac{o}{A}$ zero derivation of temperature contains heat convection, see (1)
- $\eta_1(x) = -\frac{d\lambda(x)}{dx}$ inconstant coefficient in the 1st derivation
- $\eta_2(x) = -\lambda(x)$ inconstant coefficient in the 2nd derivation
- $g > 0$ order of right side of the dif. equation
- q_j constant coefficient in the j^{th} power of polynomial right side
- $a_j(x) = \frac{x^j}{j!}$ auxiliary function for polynomial formulation

Variable x goes from 0 to length L . Inconstant coefficients $\eta_u(x)$ on the left side and right side itself have to be in polynomial form.

According to [4], the solution of (2) is in the form

$$y^{(u)}(x) = \sum_{j=0}^{m-1} y_0^{(j)} c_j^{(u)}(x) + \sum_{j=0}^g q_j b_{j+m}^{(u)}(x) \quad (3)$$

where order of derivative u goes from 0 to m . The program automatically calculates all transfer coefficients $c_j^{(u)}(x)$ and $b_{j+m}^{(u)}(x)$ for chosen point x .

As we can see from (1), thermal conductivity, heat generated and convective heat transfer coefficient are functions of variable x . Typical applications where such behavior of

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material properties can be observed are MEMS applications with Functionally Graded Material (FGM). Material properties of link element are changing in longitudinal (x) direction - see Fig. 1, according to chosen function. This functional grading can be also considered in lateral direction but material variation in this direction has to be transformed into 1D system by homogenization processes [1, 2].

Solution (3) for our case is

$$T(x) = \sum_{j=0}^1 T_0^{(j)} c_j(x) + \sum_{j=0}^g \varepsilon_j b_{j+2}(x) = c_0(x)T_0 + c_1(x)T_0' + \sum_{j=0}^g \varepsilon_j b_{j+2}(x) \quad (4)$$

where $T_0 = T(x)|_{x=0}$ and analogically for derivative T_0' .

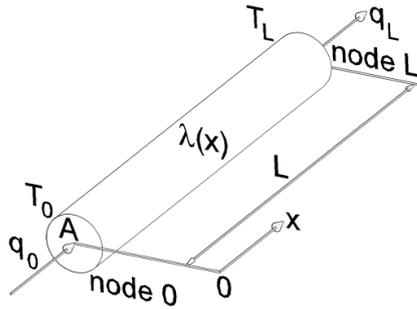


Fig. 2. Two-node link element

Let us consider two-node link element according to Fig. 2. In the first step of FEM equations derivation let us consider two boundary conditions $T(L) = T_L$ and $q(0) = q_0$. Fourier constitutive law for heat transfer for 1D problem is in the form

$$q(x) = -\lambda(x)T'(x) \quad (5)$$

Applying boundary conditions and equation (5) into (4) and after some mathematical operations we get matrix form for the 1st FEM equation

$$[c_0(L) \quad -1] \begin{bmatrix} T_0 \\ T_L \end{bmatrix} = \begin{bmatrix} \frac{c_1(L)}{\lambda_0} q_0 - \sum_{j=0}^g \varepsilon_j b_{j+2}(L) \\ 1 \end{bmatrix} \quad (6)$$

Now, let us consider $T(0) = T_0$ and $q(L) = q_L$. Applying these boundary conditions into (4) with help of (3) and (5) we get 2nd FEM equation. Then we can write both these equations in the system of FEM equations in the form

$$\begin{bmatrix} c_0(L) & -1 \\ -\left(c_0(L) - \frac{c_1(L)c_0'(L)}{c_1'(L)}\right) & 1 \end{bmatrix} \begin{bmatrix} T_0 \\ T_L \end{bmatrix} = \begin{bmatrix} \frac{c_1(L)}{\lambda_0} q_0 - \sum_{j=0}^g \varepsilon_j b_{j+2}(L) \\ \frac{c_1(L)}{c_1'(L)\lambda_L} q_L - \frac{c_1(L)}{c_1'(L)} \sum_{j=0}^g \varepsilon_j b_{j+2}'(L) + \sum_{j=0}^g \varepsilon_j b_{j+2}(L) \end{bmatrix} \quad (7)$$

The system (7) is suitable for calculation of nodal values of T and q . Next, temperature in the position x can be expressed by following equation

$$T(x) = \left[c_0(x) - c_0(L) \frac{c_1(x)}{c_1(L)} \right] T_0 + \left[\frac{c_1(x)}{c_1(L)} \right] T_L - \frac{c_1(x)}{c_1(L)} \sum_{j=0}^g \varepsilon_j b_{j+2}(L) + \sum_{j=0}^g \varepsilon_j b_{j+2}(x) \quad (8)$$

When the primary variable (temperature) is known, the secondary variables (eg. heat flux) can be calculated. Ac-

cording to [1, 2] we can express heat flux for homogenized link element $q^H(x)$. But more accurate is to determine this heat flux for the individual layers $q_k(x)$

$$q_k(x) = \frac{\lambda_k^L(x)}{\lambda^H(x)} q^H(x). \quad (9)$$

4 NUMERICAL EXPERIMENT

The thermal FGM conductor has been considered, see Fig. 3. Their square cross-section is constant with height $h = b = 6.66$ mm and length $L = 100$ mm.

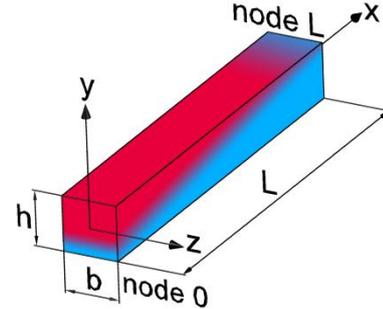


Fig. 3. FGM conductor

The FGM of this link is made by mixing two components. Material properties of the components are assumed to be constant and their values are: matrix (index m) heat conduction $\lambda_m(x, y) = 1.33$ [Wm⁻¹K⁻¹] and fibre (index f) $\lambda_f(x, y) = 450$ [Wm⁻¹K⁻¹]. Variation of the fibres volume fraction has been chosen as the polynomial function of x, y :

$$v_f(x, y) = 0.692 - 136.786x^2 + 911.905x^3 + 145y - 6.407 \cdot 10^4 x^2 y + 4.271 \cdot 10^6 x^3 y - 3750y^2 + 4.789 \cdot 10^6 x^2 y^2 - 3.193 \cdot 10^6 x^3 y^2 - 3.6 \cdot 10^6 y^3 + 1.08 \cdot 10^9 x^2 y^3 - 7.2 \cdot 10^9 x^3 y^3$$

This change of fibre volume fraction is shown in Fig. 4. There are also shown 11 functions for chosen layers used during homogenization process.

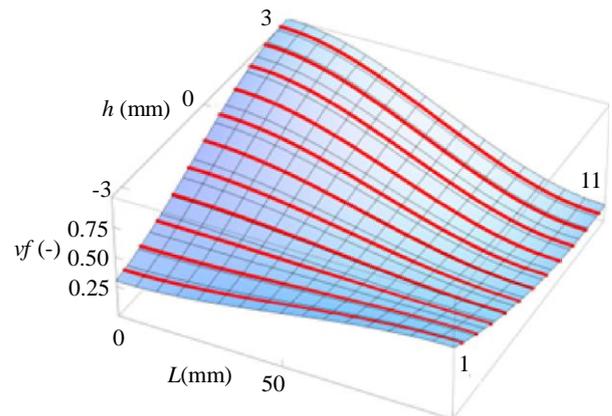


Fig. 4. Spatial 2D fibre volume fraction change

The thermal conductivity of the homogenized link has been calculated by expressions derived in [1, 2], and for $\lambda^H(x)$ we can write

$$\lambda^H(x) = \frac{2217793}{7260} - \frac{6470895}{121} x^2 + \frac{43139300}{121} x^3 \quad [\text{Wm}^{-1}\text{K}^{-1}]$$

Boundary conditions and loads are

$$t(0) = 10 \text{ }^\circ\text{C}$$

$$q(L) = -1,56 \cdot 10^4 \text{ Wm}^{-2}$$

$$Q(x) = 10^4 + 6,5 \cdot 10^7 x - 6,3 \cdot 10^9 x^3 \text{ Wm}^{-3}$$

$$\alpha(x) = 100 + 446 250x - 1,8275 \cdot 10^7 x^2 + 2,44375 \cdot 10^8 x^3 - 1,0625 \cdot 10^9 x^4 \text{ Wm}^{-2}\text{K}^{-1}$$

$$t_{ok} = 10 \text{ }^\circ\text{C}$$

They are also shown in Fig. 5.

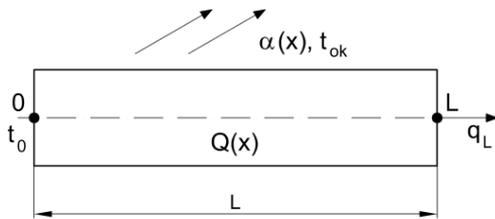


Fig. 5. Boundary conditions and loads

Form above derived equations (7-8) for our new link element, we calculated temperature for nodal values and chosen points in the field of the link element in 39 calculation points. Only one our new two-nodal finite element was used for calculation of this problem.

Because the example is relatively simple, we can use for verification purposes not only commercial FEM software ANSYS [5] but also software Mathematica [6]. In the code ANSYS the same problem has been solved using a very fine mesh – 100 000 of 3D SOLID90 elements - see Fig. 6.

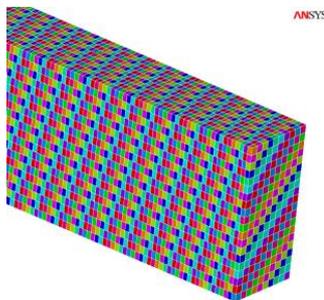


Fig. 6. Mesh of 3D elements in ANSYS

In the code Mathematica, 1D equation of heat transfer was directly solved by numerical solver of differential equations. The obtained results for temperature and heat fluxes (only layers No. 1, 6, 11 are presented) of our new approach and both verification models are shown in Fig. 7, Fig. 8 and Tab. 1. In Tab. 1 only nodal results are shown.

Tab.1. Calculated values for nodal points

	t_L [$^\circ\text{C}$]	$q_{1,0}$ [Wm^{-2}]	$q_{6,0}$ [Wm^{-2}]	$q_{11,0}$ [Wm^{-2}]	q_0^H [Wm^{-2}]
New element	12.689	-6 299.1	-13 631.7	-19 612.9	-13 361.4
ANSYS	12.672	-6 494.8	-13 875.7	-19 777.6	N/A

Mathematica	12.689	N/A	N/A	N/A	-13 361.4
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As we can see from obtained results, our new approach reached the same results as those calculated by code Mathematica. When we compare our results with ANSYS 3D element results, we can see small differences specially at the right end of link, that are caused by 3D effects modelled in code ANSYS. These effects cannot be naturally included in 1D model.

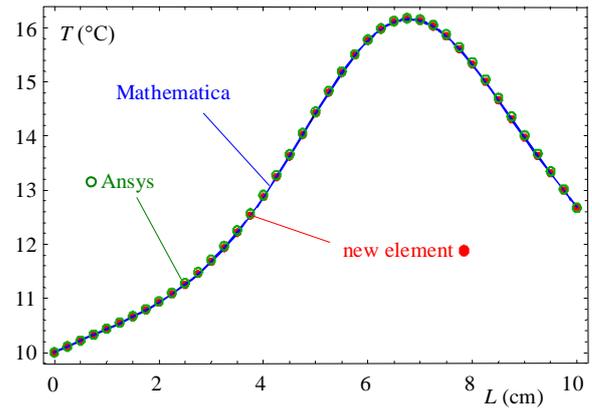


Fig. 7. Temperature in the link element

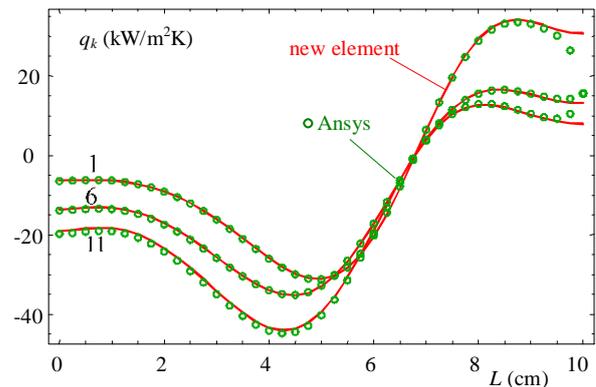


Fig. 8. Heat fluxes in the link element layers

5 CONCLUSIONS

The paper presents new approach for solution of thermal field in FGM conductors. From the results of the numerical experiment we can see an excellent agreement between the only one new 2-nodal thermal link element solution and both verification models, that were created in codes ANSYS and Mathematica. The new element solution is accurate 1D solution of the differential equation of heat transfer conduction including heat generation and convective heat transfer. As was shown in the paper, the efficiency and accuracy of this new finite element is excellent.

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